



Article

Adaptive Tracking Control for the Piezoelectric Actuated Stage Using the Krasnosel'skii-Pokrovskii Operator

Rui Xu ^{1,2} , Dapeng Tian ^{1,2,3,*} and Zhongshi Wang ^{1,2}

¹ Changchun Institute of Optics, Fine Mechanics and Physics, Chinese Academy of Sciences, Changchun 130033, China; xur@ciomp.ac.cn (R.X.); zhongshiwang@ciomp.ac.cn (Z.W.)

² Key Laboratory of Airborne Optical Imaging and Measurement, Chinese Academy of Sciences, Changchun 130033, China

³ Research Institute of Intelligent Control and Systems, Harbin Institute of Technology, Harbin 150001, China

* Correspondence: d.tian@ciomp.ac.cn

Received: 22 April 2020; Accepted: 22 May 2020; Published: 25 May 2020



Abstract: In this paper, a discrete second order linear equation with the Krasnosel'skii-Pokrovskii (KP) operator is used to describe the piezoelectric actuated stage. The weights of the KP operators are identified by the gradient descent algorithm. To suppress the hysteresis nonlinearity of the piezoelectric actuated stage, this paper proposes an adaptive tracking control with the hysteresis decomposition on the designed error surface. The proposed adaptive tracking controller dispenses with any form of the feed-forward hysteresis compensation and the unknown parameters of the discrete second order linear equation are adaptively adjusted. Some simulations are implemented to verify the effectiveness of the KP operators, then a series of modeling and control experiments are carried out on the piezoelectric actuated stages experimental systems. The comparative experimental results verify the feasibility of the KP operators modeling method and the adaptive tracking control method.

Keywords: adaptive control; hysteresis; Krasnosel'skii-Pokrovskii operator; piezoelectric actuated stage

1. Introduction

Piezoelectric actuated stages are extensively applied in the high-precision imaging mechanism and ultra-precision positioning field, such as optical alignments, scanning tunneling microscope, and photoelectric imaging tracking systems, because of their high response speed, big work-distance and strong stress [1,2]. However, the hysteresis of the piezoelectric materials damages the positioning accuracy of the stage [3]. To solve this problem, numerous modeling methods have been proposed to capture hysteresis of the piezoelectric actuated stage, which mainly include the differential equations-based models and operator-based models. So far, the Duhem model, Bouc-Wen model, and Coleman-Hodgdon model are the common differential equations-based models [4]. For example, the Coleman-Hodgdon model, which had two differential equations with different exponential, was used to describe the rising and declining hysteresis curves [5]. Lin et al. [6] conducted a depth research for a class of generalized Duhem models. Experimental results demonstrated that the generalized Duhem model can describe the hysteretic characteristics more accurately. In [7], the Bouc-Wen model was used to represent the hysteresis nonlinearity of the piezoelectric actuated stage and it got the great results. The Preisach model describes the hysteresis nonlinearity by the weighted summation of series of Preisach operators with the values of 0–1 [8–10]. The Prandtl-Ishinskii (PI) model is a modification form of Preisach model [11,12]. The PI model exploits two basic operators of Play and Stop [13]. Furthermore, it obtains widespread application for its simple mathematical

structure. For example, Kuhnen K et al. [14] investigated a modified PI model in which the different operators have different thresholds. The Krasnosel'skii-Pokrovskii (KP) model is another modification form of the Preisach model [15]. The element operator of KP model overcomes the drawback of the jump discontinuities of Preisach operators. Moreover, it includes minor loops within its major loop which make it more closed to the actual behaviors of the hysteresis nonlinearity [16,17]. In [18], a rate-dependent KP model is proposed to describe the dynamic hysteresis nonlinearity of the piezoelectric-driven platform. The unknown parameters are identified by the interior point method. The simulation results show the proposed hysteresis model can describe the hysteresis loop with the different frequencies driving input of the piezoelectric-driven platform.

Based on above hysteresis models, some control methods have been designed to eliminate the hysteresis effects and achieve high-precision trajectory tracking control for the piezoelectric actuated stage, such as the inverse feed-forward control [19], sliding mode control [20–22], and disturbance rejection control [23]. Merry et al. [24] designed a feedforward compensation controller based on an extended Coleman-Hodgdon model to eliminate the hysteresis nonlinearity. Li et al. [25] proposed an inverse compensation control method with the inverse rate-dependent PI model to solve the hysteresis of the piezoelectric actuated platform and achieve high-precision positioning control. Nguyen et al. [26] utilized a discrete-time quasi-sliding-mode control with a new sliding variable to alleviate the effects of the hysteresis nonlinearity. Shan et al. [27] proposed a robust output feedback controller to degrade the hysteresis nonlinearity of piezoelectric actuated systems, the tracking error remained less than 5%. Cao et al. [28] presented a hybrid control method by integrating the inversion feedforward control and PID-based sliding model control to compensate the hysteresis nonlinearity more effectively. Ge et al. [29] proposed a hybrid controller comprised of inverse Preisach model as feedforward compensation and PID control to improve the tracking accuracy for a piezoelectric actuated system, when the amplitude of reference displacement is 12 μm , the maximum error rate is 2.08%. While Lin et al. [30] developed an intelligent integral backstepping sliding-mode controller using a recurrent neural network to eliminate the hysteresis nonlinearity of the piezo-driven nanopositioning stage. Xu [31] utilized the dynamical model of the piezo-stage to design a digital integral terminal sliding mode controller, the effectiveness of which was supported with experimental results.

This study adopts a discrete second order linear equation with hysteresis input to express the piezoelectric actuated stage. Next, an adaptive tracking controller based on the new designed error surface is proposed to mitigate the hysteresis nonlinearity of the piezoelectric actuated stage, where the hysteresis model of this stage is separated on the proposed error surface to facilitate the design of the adaptive control approach. Comparative experimental results show that the output displacement of the piezoelectric actuated stage with the proposed adaptive controller can track the desired trajectory signal, its hysteresis characteristics have been restrained more effectively.

The remainder of this paper is organized as follows. In the Section 2, the piezoelectric actuated stage is modeled by a discrete second order linear equation with KP operators. The design of the adaptive tracking control law is presented in Section 3. The Section 4 performs the experimental setup and experimental results. Followed by the conclusion in Section 5.

2. Hysteresis Modeling of the Piezoelectric Actuated Stage

The piezoelectric actuated stage is a complex nonlinear system; it is made up of the mechanical platform and piezoelectric actuator. The mechanical platform is modeled using the mass spring damping system, and the piezoelectric actuator is regarded as an driving force with the voltage input. In general, the dynamic equation of the piezoelectric actuated stage is expressed as

$$m\ddot{y}(t) + b_1\dot{y}(t) + b_2y(t) = H_o(t), \quad (1)$$

where $y(t)$ is the output displacement of the piezoelectric actuated stage, $H_o(t)$ is the driving force from the piezoelectric actuator, and it has severe hysteresis. m , b_1 , and b_2 are the mass, friction coefficient,

and stiffness factor of the mechanical platform. In this study, we use a discrete second order linear equation to express the system (1), and its mathematical description is written as:

$$a(z^{-1})y(k) = z^{-2}b(z^{-1})H_o(k), \tag{2}$$

where the z^{-1} is the delay operator; $b(z^{-1})$ is replaced by b_0 , and $b_0 > 0$ is unknown. The $a(z^{-1})$ is expressed as:

$$a(z^{-1}) = 1 + a_1z^{-1} + a_2z^{-2}, \tag{3}$$

where the parameters a_1 and a_2 are unknown. In this paper, we use the KP operator to describe the hysteresis of the piezoelectric actuator, the $H_o(k)$ is replaced by the KP operator formula. So, the whole piezoelectric actuated stage system is expressed as:

$$\begin{cases} a(z^{-1})y(k) = z^{-2}b(z^{-1})H[u](k) \\ H[u](k) = \int \int_{\alpha \leq \beta} p(\alpha, \beta) KP_{\alpha\beta}[u](k) d\alpha d\beta, \end{cases} \tag{4}$$

where $u(k)$ is the input of the piezoelectric actuated stage at time k , respectively; $KP_{\alpha\beta}[u](k)$ is the output of the elementary KP kernel at time k , which is shown in Figure 1a. $H[u](k)$ is the elementary KP kernel with the superimposed density weight ($p(\alpha, \beta)$) [19]. $KP_{\alpha\beta}[u](k)$ is expressed as:

$$KP_{\alpha\beta}[u](k) = \begin{cases} \max[\zeta(k), r(u(k) - \beta)] & \text{if } u(k-1) < u(k) \\ \min[\zeta(k), r(u(k) - \alpha)] & \text{if } u(k-1) > u(k), \end{cases} \tag{5}$$

where α and β represent the thresholds of the KP operator. As shown in Figure 1a, it influences the width of the hysteresis loop of the KP operator. $r(\cdot)$ is the boundary auxiliary function, which is expressed as:

$$r[u](k) = \begin{cases} 0 & u(k) < 0 \\ u_{\max} * u(k) / \varrho & 0 \leq u(k) \leq \varrho \\ u_{\max} & u(k) > \varrho, \end{cases} \tag{6}$$

where ϱ is the slope of the boundary auxiliary function, it is calculated by $\varrho = (u_{\max} - u_{\min}) / (L - 1)$, u_{\max} and u_{\min} are the maximum and minimum values of the model input. Next, the Preisach plane is divided into a mesh using the L for numerically calculating the KP model in Figure 1b. For more descriptions of the Preisach plane, the reader can find it in [19]. $\zeta(k)$ is the memory value of the KP operator and it is expressed as:

$$\zeta(k) = \begin{cases} 0 & \text{for } k = 0 \\ KP_{\alpha\beta}[u](k) & \text{for } \text{sign}(\dot{u}(k)) = -\text{sign}(\dot{u}(k-1)) \\ \zeta(k-1) & \text{for } \text{sign}(\dot{u}(k)) = \text{sign}(\dot{u}(k-1)), \end{cases} \tag{7}$$

where $\text{sign}(\cdot)$ is the signum function. For ease of the algorithm design, we discrete the KP operator on Preisach plane as follows:

$$H[u](k) = \sum_{i=1}^N KP_{\alpha\beta}^i[u](k) p_i(\alpha, \beta), \tag{8}$$

where N is the number of the KP kernel, it is calculated using $N = (L + 1)(L + 2) / 2$.

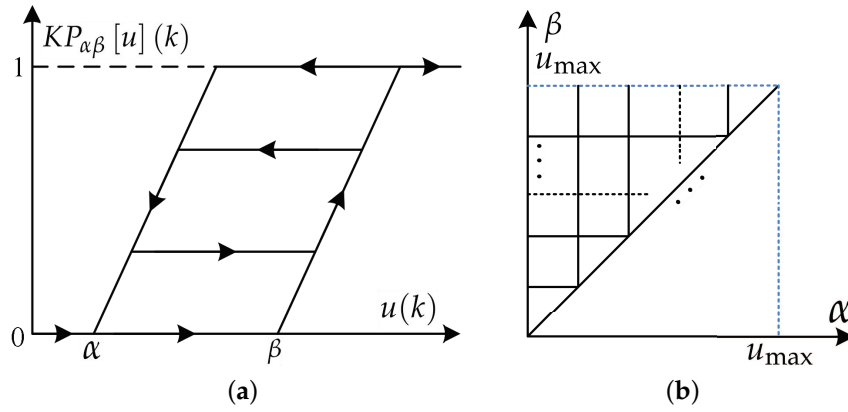


Figure 1. Structure chart of the Krasnosel'skii-Pokrovskii (KP) kernel and Preisach plane. (a) KP kernel; (b) Preisach plane.

In this study, we use the KP operator to represent the hysteresis of the piezoelectric actuated stage. So, the unknown weights must be obtained at first. Next, we use the gradient descent algorithm to identify the unknown weights of the KP operator. The identification structure diagram of the KP operators are shown in Figure 2. The input signal is used to actuate the KP kernel, then these KP kernels are superimposed with the weights to calculate the output of the KP operators. The output of the KP operator with the estimated weights is expressed as:

$$\hat{H}[u](k) = \sum_{i=1}^N KP_{\alpha\beta}^i[u](k) \hat{p}_i(\alpha, \beta), \tag{9}$$

where $\hat{p}_i(\alpha, \beta)$ is the estimated weights. Next, we set the objective function $J_g(k)$ as the sum of the squared modeling error defined by

$$J_g[u](k) = [H[u](k) - \hat{H}[u](k)]^2 / 2, \tag{10}$$

When the value of $J_g[u](k)$ is equal or small than default value, the estimated weights $\hat{p}_i(\alpha, \beta)$ are close to the desired values, the output of the KP operators is consistent with the hysteresis output of the piezoelectric actuated stage. Otherwise, it is necessary to calculate $\Delta p_i(k)$ for adjusting the weight values. The adjustment criterion is shown as:

$$\begin{aligned} \Delta p_i(k) &= \eta_g \frac{\partial J_g(k)}{\partial p_i(k)} \\ &= \eta_g (H[u](k) - \hat{H}[u](k)) KP_{\alpha\beta}^i[u](k), \end{aligned} \tag{11}$$

where η_g is the learning factor, which decides the convergence rate of the gradient descent algorithm. The unknown weights of the KP operator are updated based on the gradient descent algorithm as following:

$$p_i(k+1) = p_i(k) + \Delta p_i(k). \tag{12}$$

According to the system model (4), when the input and output of the piezoelectric actuated stage are known, the unknown parameters of the piezoelectric actuated stage model include $a_0, a_1, b_0,$ and $p(\alpha, \beta)$. In this study, the density weight $p(\alpha, \beta)$ has been obtained using the gradient descent algorithm by the offline way; other unknown parameters are adaptively acquired using the proposed adaptive control approach.

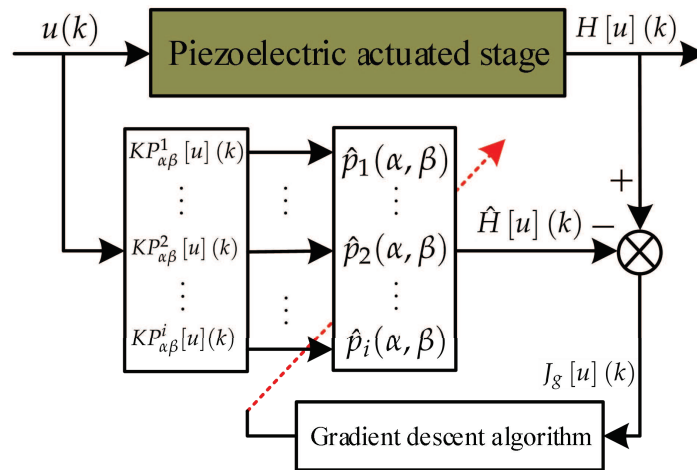


Figure 2. Identification structure diagram of the KP operators with the gradient descent algorithm.

3. Adaptive Tracking Control Law Design

The purpose of this paper is to use the proposed controller to obtain the appropriate input voltage $u(k)$, so that the output $y(k)$ of the piezoelectric actuated stage can track the desired trajectory signal $y_d(k)$ precisely. The adaptive control strategy is a control method to make the output of the nonlinear system with unknown parameters track the trajectory signal. In this section, we put forth an adaptive tracking control law based on a new design approach to mitigate the hysteresis of the piezoelectric actuated stage and achieve its high-precision tracking control. At first, the initial error surface is defined as:

$$\sigma(k+2) = c(z^{-1})(y(k+2) - y_d(k+2)), \quad (13)$$

where $c(z^{-1}) = 1 + c_1z^{-1} + c_2z^{-2}$, and the coefficient c_i is chosen such that the $c(z^{-1})$ is Hurwitz. It can be shown that, if $\lim_{k \rightarrow \infty} \sigma(k) = 0$, $\lim_{k \rightarrow \infty} (y(k) - y_d(k)) = 0$ exponentially. Since the density weights of the KP operators are obtained, we decompose the hysteresis part and the linear part of the system model on the error surface to facilitate the design of adaptive tracking controller, and the new error surface $\sigma(k+2)$ is

$$\sigma(k+2) = g(z^{-1})H[u](k) + f(z^{-1})y(k) - c(z^{-1})y_d(k+2), \quad (14)$$

where $g(z^{-1}) = g_0 + g_1z^{-1}$, $f(z^{-1}) = f_0 + f_1z^{-1}$; and the parameters g_0 , g_1 , f_0 , and f_1 can uniquely be determined in [32]. Substituting the KP operators (8) into Equation (14), one can express the error surface $\sigma(k+2)$ as:

$$\begin{aligned} \sigma(k+2) = & \phi^T(k)\theta + g_0 \sum_{i=1}^N KP_{\alpha\beta}^i[u](k)p_i(\alpha, \beta) \\ & + g_1 \sum_{i=1}^N KP_{\alpha\beta}^i[u](k-1)p_i(\alpha, \beta) - c(z^{-1})y_d(k+2), \end{aligned} \quad (15)$$

where $\phi(k) = [y(k), y(k-1)]^T$, $\theta = [f_0, f_1]^T$. For now, we assume that all of the parameters are known. According to Equation (15), we get that the input $u(k)$ satisfies

$$\begin{aligned} g_0 \sum_{i=1}^N KP_{\alpha\beta}^i[u](k)p_i(\alpha, \beta) = & -g_1 \sum_{i=1}^N KP_{\alpha\beta}^i[u](k-1)p_i(\alpha, \beta) \\ & - \phi^T(k)\theta + c(z^{-1})y_d(k+2), \end{aligned} \quad (16)$$

the error surface $\sigma(k + 2) = 0$, the actual output of the piezoelectric actuated stage is consistent with the desired trajectory signal. Next, we write the estimated values of the parameters as $\hat{g}(k) = [\hat{g}_0(k), \hat{g}_1(k)]^T$, $\hat{\theta}(k) = [\hat{f}_0(k), \hat{f}_1(k)]^T$. The unknown parameters on the right side of the Equation (15) are replaced by the estimated values, we define $\bar{\sigma}(k)$ as $\bar{\sigma}(k) = \sigma(k) + c(z^{-1})y_d(k)$. Then, the estimation error is expressed as:

$$e(k) = \bar{\sigma}(k) - \phi^T(k-2)\hat{\theta}(k-1) - \hat{g}_0 \sum_{i=1}^N KP_{\alpha\beta}^i[u](k)p_i(\alpha, \beta) - \hat{g}_1 \sum_{i=1}^N KP_{\alpha\beta}^i[u](k-1)p_i(\alpha, \beta), \tag{17}$$

Substituting Equation (15) into Equation (17), the estimated error $e(k)$ can be rewritten as:

$$e(k) = -y(k-2) \cdot \tilde{f}_0(k-1) - y(k-3) \cdot \tilde{f}_1(k-1) - \tilde{g}_0 \sum_{i=1}^N KP_{\alpha\beta}^i[u](k)p_i(\alpha, \beta) - \tilde{g}_1 \sum_{i=1}^N KP_{\alpha\beta}^i[u](k-1)p_i(\alpha, \beta), \tag{18}$$

where the new variables are $\tilde{f}_i(k) = \hat{f}_i(k) - f_i(k)$, (i.e., $\tilde{\theta}(k)$), $\tilde{g}_i(k) = \hat{g}_i(k) - g_i(k)$, ($i = 0, 1$). To estimate unknown parameters of the linear part, the cost function is set as:

$$J(f_i, g_i)(k) = \frac{1}{2}e^2(k). \tag{19}$$

Next, we adopt the following Equation (20) to adjust unknown parameters adaptively.

$$\begin{aligned} \dot{\hat{\theta}}(k) &= -\gamma \frac{\partial J(k)}{\partial \hat{\theta}(k)} = -\gamma e(k) \frac{\partial e(k)}{\partial \hat{\theta}(k)} \\ \dot{\hat{g}}_i(k) &= -\gamma \frac{\partial J(k)}{\partial \hat{g}_i(k)} = -\gamma e(k) \frac{\partial e(k)}{\partial \hat{g}_i(k)}, \end{aligned} \tag{20}$$

where γ is the adjustment speed of the estimated parameters, and it satisfies $0 < \gamma < 1$. If a fast adaptation of the unknown parameters is needed, the γ should be increased in the range $0 < \gamma < 1$. Substituting Equation (18) into Equation (20), the adjustment results for each parameter can be derived as follows:

$$\begin{aligned} \hat{f}_0(k) &= \hat{f}_0(k-1) - \gamma \cdot e(k) \cdot y(k-2), \\ \hat{f}_1(k) &= \hat{f}_1(k-1) - \gamma \cdot e(k) \cdot y(k-3). \end{aligned} \tag{21}$$

$$\begin{aligned} \hat{g}_0(k) &= \hat{g}_0(k-1) - \gamma \cdot e(k) \cdot \sum_{i=1}^N KP_{\alpha\beta}^i[u](k)p_i(\alpha, \beta), \\ \hat{g}_1(k) &= \hat{g}_1(k-1) - \gamma \cdot e(k) \cdot \sum_{i=1}^N KP_{\alpha\beta}^i[u](k-1)p_i(\alpha, \beta). \end{aligned} \tag{22}$$

The unknown parameters are estimated by Equation (21) and Equation (22). The convergent of the parameters estimated method is not proved in this paper, the more discussions of this method, the reader may refer to [33]. Next, we propose the adaptive control law (23) to eliminate the hysteresis and achieve the high-precision trajectory tracking control for the piezoelectric actuated stage.

$$u(k+1) = u(k) - \eta \cdot \left(\psi(k) - \hat{g}_0 \sum_{i=1}^N KP_{\alpha\beta}^i[u](k)p_i(\alpha, \beta) \right), \tag{23}$$

where the constant η influences the adjustment speed of the input voltage, $\psi(k)$ is expressed as:

$$\psi(k) = -\phi^T \hat{\theta}(k) - \hat{g}_1 \sum_{i=1}^N KP_{\alpha\beta}^i[u](k-1)p_i(\alpha, \beta) + c(z^{-1})y_d(k+2) + \kappa\sigma(k), \tag{24}$$

where κ is the weight factor. Based on the Equation (16), if $\psi(k)$ satisfies the following equation:

$$\psi(k) = \sum_{i=1}^N KP_{\alpha\beta}^i[u^*](k)p_i(\alpha, \beta), \tag{25}$$

where $u^*(k)$ is equal to the ideal input voltage, and the $\sigma(k)$ will tend to zero. However, the actual input voltage $u^*(k)$ cannot be accurately calculated. This paper use a weakened condition of Equation (25): it satisfies that $|\sum_{i=1}^N KP_{\alpha\beta}^i[u^*](k)p_i(\alpha, \beta) - \psi(k)| \leq \varepsilon$, for a given admissible error bound ε . So, the adaptive tracking control law (23) can make the tracking error $e(k)$ be approximated to a tolerable error range at the $k \rightarrow \infty$.

4. Simulation and Experimental Results

4.1. Simulation Results

In this part, we conducted some simulation studies of the KP kernel with the weights to describe the hysteresis in the piezoelectric actuated stage. In simulation, we used three KP kernel with the different weights (which were 0.2, 0.6, and 0.3) as the actual system, then the gradient descent algorithm was used to identify the unknown weights for verifying the validity of the proposed KP operator modeling method. The simulation results are shown in Figures 3 and 4. When the input signal was the triangular wave, it is obvious that the modeling error was close to zero after a period. To avoid the contingently of the simulation results, the sine wave signal was used as the input signal; the output of the KP operators based on the gradient descent algorithm was consistent with the output of actual system after a period from the Figure 4. Simulation results showed that the proposed KP operators based on the gradient descent algorithm could describe the hysteresis of the piezoelectric actuated stage.

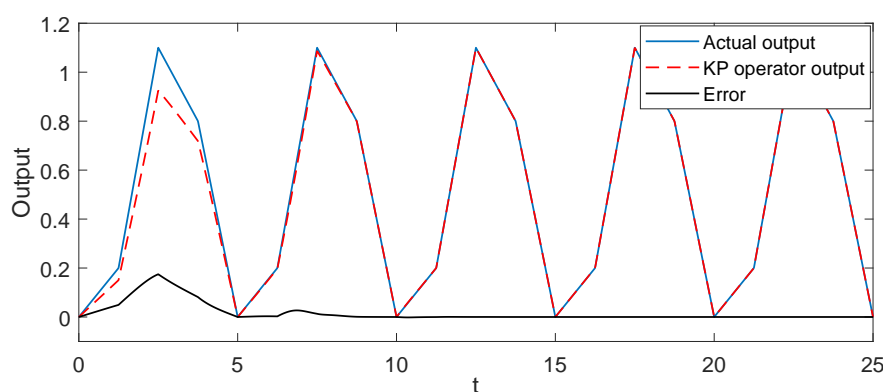


Figure 3. Simulation results of the KP model with the gradient descent algorithm under the triangular wave voltage signal.

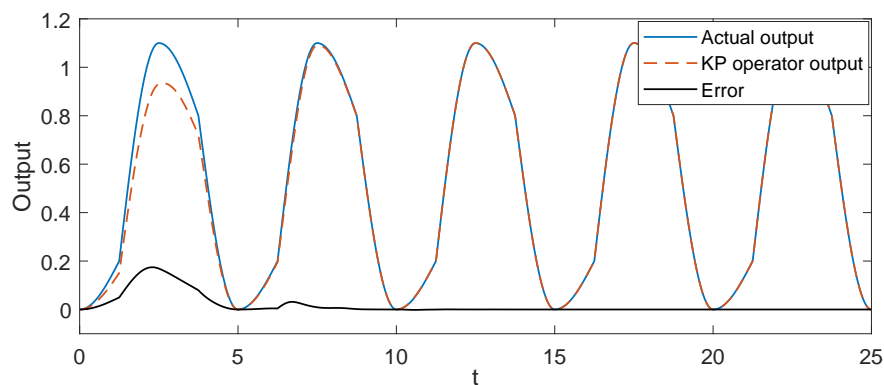


Figure 4. Simulation results of the KP model with the gradient descent algorithm under the sine wave voltage signal.

4.2. Experimental Results

In this part, an experimental facility, as shown in Figure 5, was set up to verify the effectiveness of the proposed adaptive tracking control method. It consisted of the piezoelectric actuated stage (MPT-2MRL102A, HuiBo Robotics Technology Co., Ltd, China), the computer, the data collection card (PCI-1710), and the integral driving controller. In the experiment, the control algorithm was implemented using the Matlab/simulink software in the host computer. The integral driving controller with the open-loop mode drove the piezoelectric actuated stage to generate the output displacement. The data collection card was used to handle the D/A or A/D conversion and collect output displacement data of the piezoelectric actuated stage.

At first, we set triangular wave signal as the driven signal to actuate the piezoelectric actuated stage. The learning factor η_g was set as 0.01; the cutting line of the Preisach plane L was set as 17 by trial and error in this paper. The identification results are shown in Figure 6. It notes that a larger modeling error was generated as the first period, and then the modeling error decreased gradually to stabilization. The maximum modeling steady error was $0.53 \mu\text{m}$, and the corresponding relative error rate was 0.99%. So, the KP operators with the obtained weights could capture the hysteresis characteristics of the piezoelectric actuated stage.

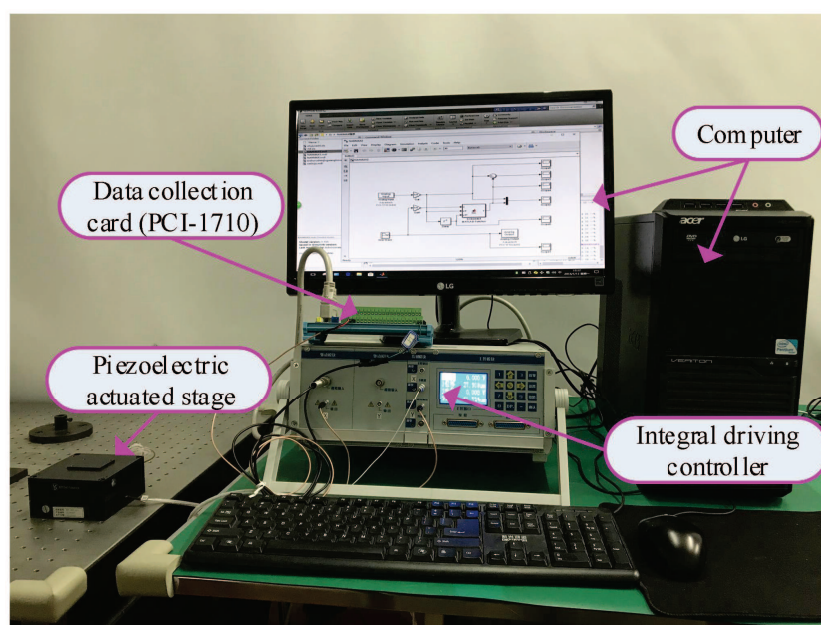


Figure 5. Picture of the piezoelectric actuated stage experimental setup.

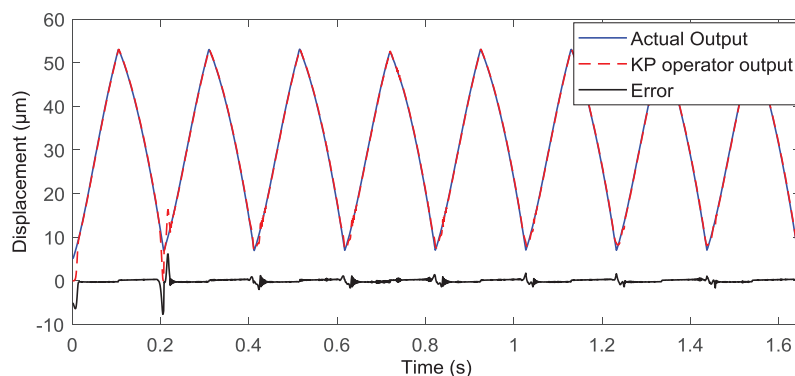


Figure 6. Experimental result of KP operator based on the gradient descent algorithm under the triangular wave voltage signal.

To avoid the contingency of modeling experimental results, the complex wave signal (i.e., $u = 22.5 \sin(10\pi t - 0.5\pi) + 15 \sin(2\pi t - 0.5\pi) + 37.5$) was the input voltage signal to actuate the piezoelectric actuated stage. The modeling experimental results are shown in Figure 7, the output of the KP operators with the obtained weights were consistent with the actual output of the piezoelectric actuated stage. The maximum modeling steady error rate was 1.21%; it satisfies the requirements of the practical engineering application.

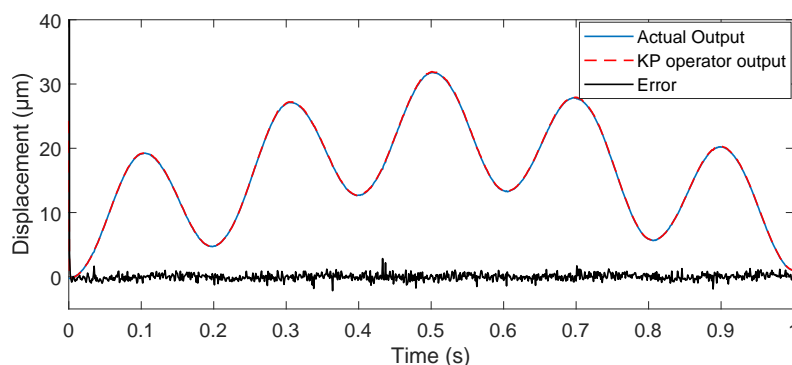


Figure 7. Experimental result of KP operator based on the gradient descent algorithm under the complex wave signal.

To prove the effectiveness of the proposed adaptive control method, the triangular wave signal was used as the desired trajectory signal to implement the tracking control experiments. The parameters of the proposed controller were set as $\gamma = 0.2$, $\eta = 0.025$, and $\kappa = 6.8$. It can be seen from Figure 8, the estimated unknown parameters could converge to their true values quickly. The convergence time of the proposed adaptive controller was about 0.25 s. Figure 9 shows that the output displacement of the piezoelectric actuated stage based on the proposed adaptive tracking controller was consistent with the desired trajectory signal. The mean tracking error and root-mean-square error of the controller were $0.2526 \mu\text{m}$ and $0.3063 \mu\text{m}$, respectively. The effectiveness of the proposed adaptive controller was verified.

Furthermore, in order to further validate the superior performance of the proposed adaptive controller, we chose the complex wave signal as the desired trajectory signal to carry out the comparison tracking experiments. The parameters of the proposed controller were the same as above. The experimental results of the proposed adaptive tracking control method for the piezoelectric actuated stage are shown in Figures 10 and 11. Figure 10 showed that the estimated parameters (\hat{f}_0 , \hat{f}_1 , \hat{g}_0 , and \hat{g}_1) could converge quickly to 2.2782, -0.2148 , 0.0723, and 0.0355. Figure 11 showed that the proposed adaptive control method achieved more accurate tracking results in comparison with the

traditional sliding mode control (SMC) method, where the mean tracking error was reduced by 23.43%. It is concluded from experimental results that the proposed adaptive tracking controller is feasible.

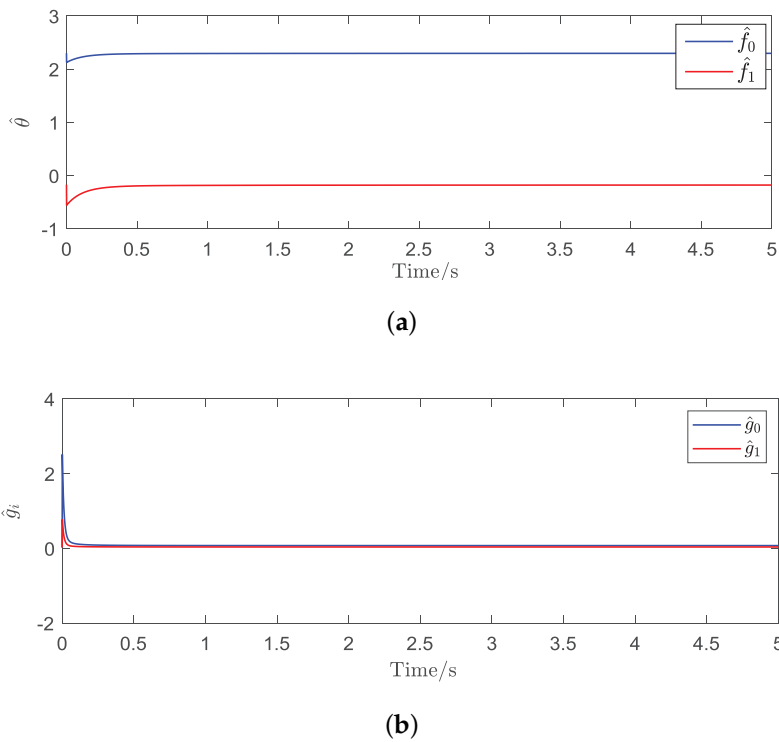


Figure 8. Estimated parameters $\hat{\theta}$ and \hat{g}_i of the proposed adaptive controller for the triangular wave signal. (a) convergent curve of $\hat{\theta}$; (b) convergent curve of \hat{g}_i .

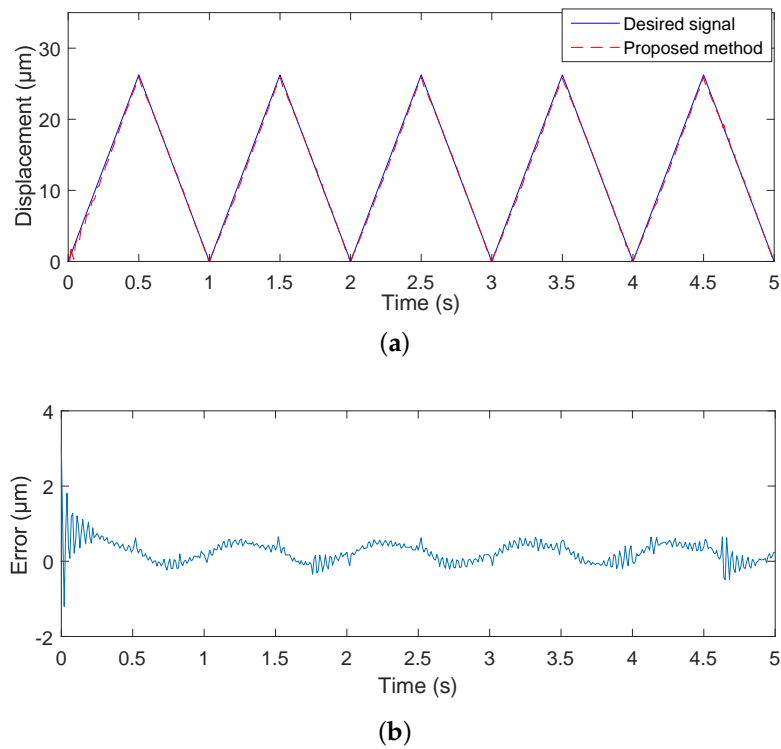
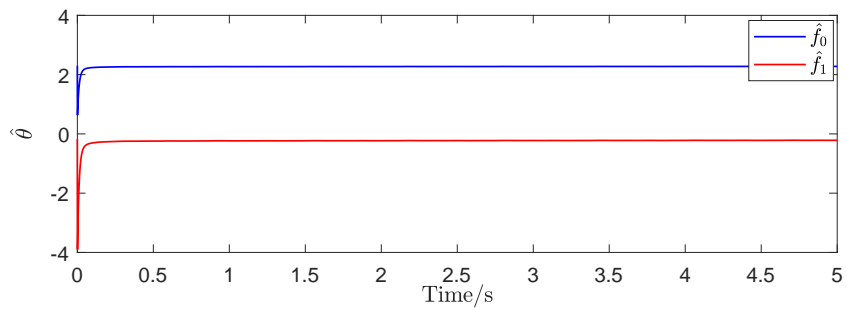
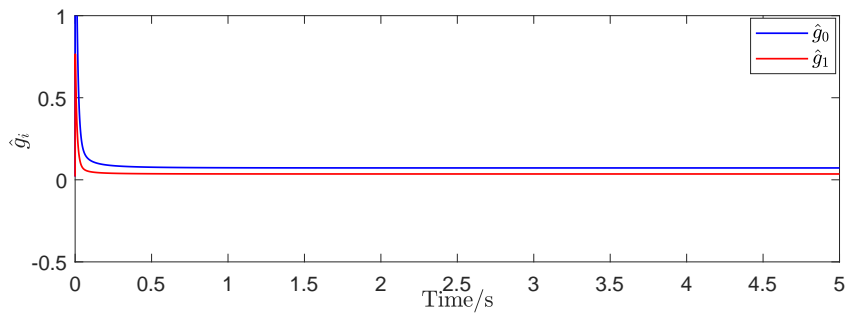


Figure 9. Experimental results of the piezoelectric actuated stage with the triangular wave signal. (a) tracking curve of the piezoelectric actuated stage; (b) tracking error.

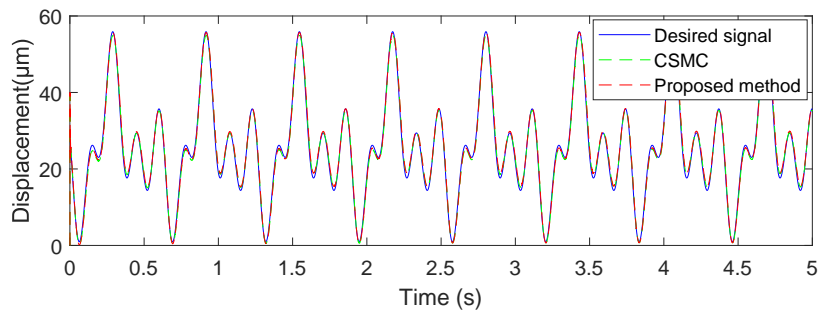


(a)

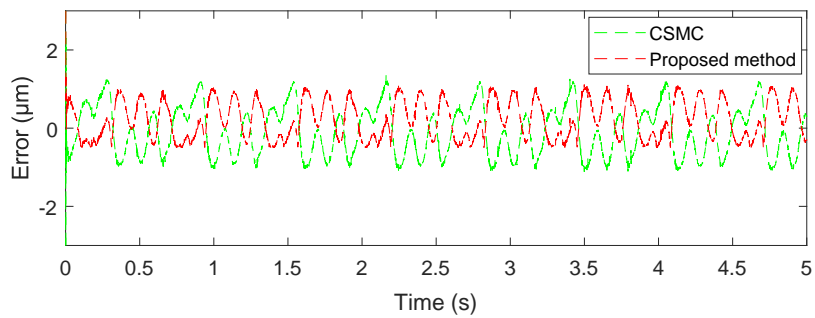


(b)

Figure 10. Estimates results of the parameters $\hat{\theta}$ and \hat{g}_i . (a) convergent curve of $\hat{\theta}$; (b) convergent curve of \hat{g}_i .



(a)



(b)

Figure 11. Comparative results of the piezoelectric actuated stage with the complex wave signal. (a) tracking curve of the piezoelectric actuated stage; (b) tracking error.

5. Conclusions

In this paper, the KP operator with the weights identified by the gradient descent algorithm is used to capture the hysteresis part of the piezoelectric actuated stage. Then, the discrete second order linear equation with the KP operators describes the whole piezoelectric actuated stage system. Next, an adaptive tracking control methodology has been designed for the piezoelectric actuated stage to track the desired trajectory signal. The unknown parameters of the stage systems are adaptively obtained and converge to their desired values. Some comparative tracking experiments are implemented on the piezoelectric actuated stage, where the results verify the superior performance of the proposed adaptive controller in tracking the desired complex wave signal compared with the traditional SMC method.

Author Contributions: Conceptualization, R.X.; methodology, R.X. and Z.W.; validation, Z.W. and D.T.; formal analysis: R.X. and D.T.; writing—original draft preparation, R.X. and D.T.; writing—review and editing: R.X. and D.T.; supervision and project administration, D.T. All authors have read and agreed to the published version of the manuscript.

Funding: This work was supported in part by National Natural Science Foundation of China (grant 61673365); Key Research Program of Frontier Sciences, CAS (grant ZDBS-LY-JSC044); The Youth Innovation Promotion Association of Chinese Academy of Sciences (grant 2017257).

Conflicts of Interest: The authors declare no conflict of interest.

References

- Zawiarta, M.; Fleming, A.; Putrino, G.; Yong, Y. Multimodal atomic force microscopy with optimized higher eigenmode sensitivity using on-chip piezoelectric actuation and sensing. *Nanotechnology* **2019**, *8*, 085503.
- Gu, G.Y.; Zhu, L.M.; Su, C.Y. Modeling and compensation of asymmetric hysteresis nonlinearity for piezoceramic actuators with a modified Prandtl-Ishlinskii model. *IEEE Trans. Ind. Electron.* **2014**, *61*, 1583–1595. [[CrossRef](#)]
- Qin, Y.D.; Duan, H. Single-Neuron adaptive hysteresis compensation of piezoelectric actuator based on hebb learning rules. *Micromachines* **2020**, *11*, 84. [[CrossRef](#)] [[PubMed](#)]
- Ding, B.X.; Li, Y.M. Hysteresis compensation and sliding mode control with perturbation estimation for piezoelectric actuators. *Micromachines* **2018**, *9*, 241. [[CrossRef](#)]
- Ounissi, A.; Landry, M.; Kaddouri, A.; Abdessemed, R. PSO based parameter identification of Colman-Hodgdon hysteresis model of a piezoelectric actuator and PID feedback controller. *Contemp. Eng. Sci.* **2014**, *7*, 179–192. [[CrossRef](#)]
- Lin, C.J.; Lin, P.T. Tracking control of a biaxial piezo-actuated positioning stage using generalized Duhem model. *Comput. Math. Appl.* **2012**, *64*, 766–787. [[CrossRef](#)]
- Xu, R.; Zhang, X.Y.; Guo, H.Y.; Zhou, M.L. Sliding mode tracking control with perturbation estimation for hysteresis nonlinearity of piezo-actuated stages. *IEEE Access* **2018**, *6*, 30617–30629. [[CrossRef](#)]
- Li, Z.; Su, C.Y.; Chai, T.Y. Compensation of hysteresis nonlinearity in magnetostrictive actuators with inverse multiplicative structure for Preisach model. *IEEE Trans. Autom. Sci. Eng.* **2014**, *11*, 613–619. [[CrossRef](#)]
- Li, C.T.; Tan, Y.H. Adaptive output feedback control of systems preceded by the Preisach-type hysteresis. *IEEE Trans. Syst. Man Cybern. B Cybern.* **2005**, *35*, 130–135. [[CrossRef](#)]
- Xiao, S.L.; Li, Y.M. Modeling and high dynamic compensating the rate-dependent hysteresis of piezoelectric actuators via a novel modified inverse Preisach model. *IEEE Trans. Control Syst. Technol.* **2013**, *21*, 1549–1557. [[CrossRef](#)]
- Liu, S.N.; Su, C.Y.; Li, Z. Robust adaptive inverse control of a class of nonlinear systems with Prandtl-Ishlinskii hysteresis model. *IEEE Trans. Autom. Control.* **2014**, *59*, 2170–2175. [[CrossRef](#)]
- Wang, Y.F.; Xu, R.; Zhou, M.L. Prandtl-Ishlinskii modeling for giant magnetostrictive actuator based on internal time-delay recurrent neural network. *IEEE Trans. Magn.* **2018**, *54*, 7301004.
- Janaideh, M.A.; El-Shaer, A.H. Performance enhancement for a class of hysteresis nonlinearities using disturbance observers. *Int. J. Control Autom. Syst.* **2014**, *12*, 283–293. [[CrossRef](#)]
- Kuhnen, K. Modeling, identification and compensation of complex hysteretic nonlinearities: A modified Prandtl-Ishlinskii approach. *Eur. J. Control* **2003**, *9*, 407–418. [[CrossRef](#)]

15. Xu, R.; Zhou, M.L. Elman neural network-based identification of Krasnosel'skii-Pokrovskii model for magnetic shape memory alloys actuator. *IEEE Trans. Magn.* **2017**, *53*, 2002004. [[CrossRef](#)]
16. Zakerzadeh, M.R.; Sayyaadi, H.; Zanjani, M.A.V. Characterizing hysteresis nonlinearity behavior of SMA actuators by KP model. *Appl. Math.* **2011**, *1*, 28–38. [[CrossRef](#)]
17. Wang, Y.; Su, C.Y.; Hong, H. Model reference control including adaptive inverse hysteresis for systems with unknown input hysteresis. In Proceedings of the 2007 IEEE International Conference on Networking, Sensing and Control, London, UK, 15–17 April 2007; pp. 70–75.
18. Pan, W.; Xu, R.; Zhou, M.L. Modeling of hysteresis for piezo-driven stages using a rate-dependent Krasnosel'skii-Pokrovskii model. In Proceedings of the 2019 Chinese Control Conference (CCC2019), Guangzhou, China, 27–31 July 2019; pp. 1604–1609.
19. Li, Z.; Shan, J.J.; Gabbert, U. Inverse compensation of hysteresis using Krasnoselskii-Pokrovskii model. *IEEE/ASME Trans. Mechatron.* **2018**, *2*, 966–971. [[CrossRef](#)]
20. Zhang, Y.; Yan, P. Adaptive observer-based integral sliding mode control of a piezoelectric nano-manipulator. *IET Control Theory Appl.* **2019**, *14*, 2173–2180. [[CrossRef](#)]
21. Xu, R.; Zhou, M.L. Sliding mode control with sigmoid function for the motion tracking control of the piezo-actuated stages. *Electron. Lett.* **2017**, *53*, 75–77. [[CrossRef](#)]
22. Kim, I.; Kim, Y.S.; Park, E.C. Sliding mode control of the inchworm displacement with hysteresis compensation. *Int. J. Precis. Eng. Man* **2009**, *3*, 43–49. [[CrossRef](#)]
23. Nie, Z.; Ma, Y.; Liu, R.; Guo, D. Improved disturbance rejection control for piezoelectric actuators based on combination of ESO and Q-filter. *Electron. Lett.* **2018**, *54*, 872–874.
24. Merry, R.; Uyanik, M.; Van, D.M.R.; Koops, R.; Veghel, M.V.; Steinbuch, M. Identification control and hysteresis compensation of a 3 DOF metrological AFM. *Asian J. Control* **2009**, *11*, 130–143. [[CrossRef](#)]
25. Yang, M.J.; Li, C.X.; Gu, G.Y.; Zhu, L.M. Modeling and compensating the dynamic hysteresis of piezoelectric actuators via a modified rate-dependent Prandtl-Ishlinskii model. *Smart Mater. Struct.* **2015**, *12*. [[CrossRef](#)]
26. Nguyen, M.L.; Chen, X.K.; Yang, F. Discrete-time quasi-sliding-mode control with prescribed performance function and its application to piezo-actuated positioning systems. *IEEE Tans. Ind. Electron.* **2018**, *65*, 942–950. [[CrossRef](#)]
27. Shan, J.J.; Yang, L.; Li, Z. Output feedback integral control for nano-positioning using piezoelectric actuators. *Smart Mater. Struct.* **2015**, *24*, 045001. [[CrossRef](#)]
28. Cao, Y.; Chen, X.B. Integrated inversion-feedforward and PID-based-sliding-mode-control for piezoelectric actuators. In Proceedings of the 2012 American Control Conference (ACC), Montreal, QC, Canada, 27–29 June 2012; pp. 869–874.
29. Ge, P.; Jouaneh, M. Generalized Preisach model for hysteresis nonlinearity of piezoceramic actuators. *Precis. Eng.* **1977**, *20*, 99–111. [[CrossRef](#)]
30. Lin, F.J.; Lee, S.Y. Intelligent integral backstepping sliding-mode control for piezo-flexural nanopositioning stage. In Proceedings of the 2015 IEEE 2nd International Future Energy Electronics Conference (IFEEEC), Taipei, Taiwan, 1–4 November 2015; pp. 1–6.
31. Xu, Q.S. Digital integral terminal sliding mode predictive control of piezoelectric-driven motion system. *IEEE Trans. Ind. Electron.* **2016**, *6*, 3976–3984. [[CrossRef](#)]
32. Goodwin, G.C.; Sin, K.S. *Adaptive Filtering, Prediction and Control*; Prentice-Hall: Englewood Cliffs, NJ, USA, 1984.
33. Ioannou, P.A.; Sun, J. *Robust Adaptive Control*; Prentice-Hall: Englewood Cliffs, NJ, USA, 1995.

